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UTILITY **PATENT APPLICATION TRANSMITTAL**

Attorney Docket No. 99B140 First Inventor or Application Identifier Antulio TARZONA Title | Improvements In Valves

(Only for new nonprovisional applications under 37 C.F.R. § 1.53(b)) Express	Mail Label No. EF 132182188US
APPLICATION ELEMENTS See MPEP chapter 600 concerning utility patent application contents.	Assistant Commissioner for Patents ADDRESS TO: Box Patent Application Washington, DC 20231
1. X * Fee Transmittal Form (e.g., PTO/SB/17) (Submit an original and a duplicate for fee processing) 2. X Specification [Total Pages 9] - Descriptive title of the Invention - Cross References to Related Applications - Statement Regarding Fed sponsored R & D - Reference to Microfiche Appendix	5. Microfiche Computer Program (Appendix) 6. Nucleotide and/or Amino Acid Sequence Submission (if applicable, all necessary) a. Computer Readable Copy b. Paper Copy (identical to computer copy) c. Statement verifying identity of above copies
- Background of the Invention	ACCOMPANYING APPLICATION PARTS
- Brief Summary of the Invention - Brief Description of the Drawings (if filed) - Detailed Description - Claim(s) - Abstract of the Disclosure 3. X Drawing(s) (35 U.S.C. 113) [Total Sheets 1] 4. Oath or Declaration [Total Pages] a. Newly executed (original or copy) b. Copy from a prior application (37 C.F.R. § 1.63(d)) (for continuation/divisional with Box 16 completed) i. DELETION OF INVENTOR(S) Signed statement attached deleting inventor(s) named in the prior application, see 37 C.F.R. § 1.63(d)(2) and 1.33(b). *NOTE FOR ITEMS 1 & 13 IN ORDER TO BE ENTITIED TO PAY SMALL ENTITY FEES, A SMALL ENTITY STATEMENT IS REQUIRED (37 C.F.R. § 1.27), EXCEPT IF ONE FILED IN A PRIOR APPLICATION IS RELIED UPON (37 C.F.R. § 1.28).	7. Assignment Papers (cover sheet & document(s)) 8. 37 C.F.R.§3.73(b) Statement Power of (when there is an assignee) 9. English Translation Document (if applicable) 10. Information Disclosure Copies of IDS Statement (IDS)/PTO-1449 Citations 11. Preliminary Amendment 12. X Return Receipt Postcard (MPEP 503) (Should be specifically itemized) * Small Entity Statement(s) Statement filed in prior application Status still proper and desired (PTO/SB/09-12) 14. X (if foreign priority is claimed) 15. Other:
16. If a CONTINUING APPLICATION, check appropriate box, and supp Continuation Divisional Continuation-in-part (CIP) Prior application information: Examiner For CONTINUATION or DIVISIONAL APPS only: The entire disclosure of tunder Box 4b, is considered a part of the disclosure of the accompanying reference. The incorporation can only be relied upon when a portion has	of prior application No/ Group / Art Unit: the prior application, from which an oath or declaration is supplied a continuation or divisional application and is hereby incorporated by
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TOTAL AMOUNT OF PAYMENT

(\$) 710.60

Complete if Known					
Application Number					
Filing Date					
First Named Inventor	Antulio TARAZONA				
ExaminerName	N/A				
Group / Art Unit	N/A				
Attorney Docket No.	99B140				

METHOD OF PAYMENT (check one)	FEE CALCULATION (continued)					
1. X The Commissioner is hereby authorized to charge indicated fees and credit any over payments to:	3. ADDITIONAL FEES Large Entity Small Entity Fee	Fee Paid				
Deposit Account 02-2865	Code (\$) Code (\$) ree Description 105 130 205 65 Surcharge-late filing fee or oath					
Number Deposit Account Name The BOC Group, Inc.	127 50 227 25 Surcharge-late provisional filing fee or cover sheet.	0.00				
X Charge Any Additional	139 130 139 130 Non-English specification	0.00				
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2. Payment Enclosed:	112 920* 112 920* Requesting publication of SIR prior to Examiner action	0.00				
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FEE CALCULATION	115 110 215 55 Extension for reply within first month	0.00				
1. BASIC FILING FEE	116 380 216 190 Extension for reply within second month	0.00				
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106 310 206 155 Design filing fee 710.00	119 300 219 150 Notice of Appeal	0.00				
107 480 207 240 Plantfiling fee	120 300 220 150 Filing a brief in support of an appeal	0.00				
108 760 208 380 Reissue filing fee	121 260 221 130 Request for oral hearing	0.00				
114 150 214 75 Provisional filing fee	138 1,510 138 1,510 Petition to institute a public use proceeding	0.00				
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2. EXTRA CLAIM FEES Fee from	142 1,210 242 605 Utility issue fee (orreissue)	0.00				
Ex <u>tra Claims below</u> Fee Paid	143 430 243 215 Design issue fee	0.00				
Total Claims 11 -20** = 0	144 580 244 290 Plantissuefee	0.00				
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**or number previously paid, if greater; For Reissues, see below	123 50 123 50 Petitions related to provisional applications	0.00				
Large Entity Small Entity	126 240 126 240 Submission of Information Disclosure Stmt	0.00				
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103 18 203 9 Claims in excess of 20	property (times number of properties) 146 760 246 380 Filing a submission after final rejection	0.00				
102 78 202 39 Independent claims in excess of 3	(37 CFR 1.129(a)) 149 760 249 380 Foreach additional invention to be	0.00				
104 260 204 130 Multiple dependent claim, if not paid	examined (37 CFR 1.129(b))	0.00				
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SUBMITTED B	Υ	Complete (ifapplicable)			
Typed or Printed Name	Philip M. You Neida			Pea Number	34,942
Signature	16/11/-20	Date	10/24/00	Deposit Account User ID	02-2865

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IN THE UNITED STATES PATENT AND TRADEMARK OFFICE

NEW PATENT APPLICATION

IMPROVEMENTS IN VALVES

Inventor (s):

TARAZONA, Antulio

SMITH, John Cambridge

CURRINGTON, lan

IMPROVEMENTS IN VALVES

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FIELD OF THE INVENTION

The present invention relates to valves and in particular to pressure relief valves.

BACKGROUND OF THE INVENTION

It is known, for example, to locate a pressure relief valve between high and low pressure regions in a pneumatic or vacuum system such valves are actuated at a predetermined differential pressure to relieve the high pressure either to the low pressure region or to atmosphere. This known type of pressure relief valve uses either a spring or sometimes gravity alone to bias a valve stem towards a cooperating valve seat to maintain the valve in a normally closed position.

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When this type of valve is required to operate free of oscillations it is common practice to incorporate a damping mechanism. The damping mechanism requires components which are manufactured to close tolerances and in environments where condensation and/or solid deposition may be formed within the valve this presents a risk of malfunction particularly with a spring-biased mechanism.

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SUMMARY OF THE INVENTION

It is an aim of the present invention to provide a valve which incorporates a magnetic means for biasing the valve towards its closed position and which operates substantially free of oscillations while maintaining large internal clearances. This permits the valve to operate in environments where condensation and/or solid deposition may occur.

According to the present invention, a valve comprises a housing having an inlet and spaced therefrom an outlet, a passageway extending between the inlet and the outlet and means located in the passageway for controlling the flow of a fluid between the inlet and the outlet, the means including a valve assembly movable between a first open position spaced from a co-operating valve seat and a second closed position at which the valve assembly sealingly engages the valve seat, in which magnetic means is provided for biasing the valve assembly towards the second closed position.

In a preferred embodiment, at least a portion of the valve assembly is in the form of or incorporates a permanent magnet and a further magnet is located adjacent the valve seat. The further magnet may be a permanent magnet or alternatively an electro-magnet.

BRIEF DESCRIPTION OF THE DRAWINGS

An embodiment of the invention will now be described, by way of example only, reference being made to the accompanying drawing which is a cross-section of a pressure relief valve according to the present invention.

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DETAILED DESCRIPTION OF THE INVENTION

As shown, a pressure relief valve 1 includes a housing 2 having an inlet 4 and spaced therefrom an outlet 6. Located in the passageway extending between the inlet 4 and the outlet 6 is means including a valve assembly 8 and a co-operating valve seat 10 for controlling the flow of a fluid, for example a gas between the inlet 4 and said outlet 6.

The valve assembly 8 depends from a valve cap 9 of magnetic material which is sealingly engaged in the upper (as shown) end of the housing 2. Surrounding that portion of the valve assembly 8 within the valve cap 9 is a polymer bush 7.

The valve stem assembly 8 includes a valve stem 12 from which extends radially outwardly therefrom a recessed flange 17. Resting on the upper surface of the recessed flange 17 and surrounding the valve stem 12 are a polymer shock absorber 11 and a magnetic stainless steel washer 13.

Attached to the lower (as shown) end of the recessed flange 17 by means of a fastener 18 is a spherical seal pad 15 and located within the recessed flange 17 above the spherical seal pad 15 is a permanent magnet 14 mounted against a mounting aid 5.

The valve seat 10 is made from magnetic material and adjacent the valve seat on that side of the valve seat opposite the spherical seal pad 15 is a magnet 16 also mounting against a mounting aid 5. The magnet 16 may be in the form of a permanent magnet or alternatively an electro-magnet.

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As shown, a magnetic stainless steel sleeve 3 depends from the valve cap 9 and surrounds the valve assembly 8.

The pressure relief valve 1 operates between two stable positions, namely fully shut and fully open. With the valve 1 in its fully shut position as shown, fluid is prevented from flowing through the passageway between the inlet 4 and the outlet 6 of the housing 2 since the spherical seal pad 15 sealingly engages the valve seat 10. The valve 1 will remain shut until the pressure differential between the inlet 4 and the outlet 6 increases to such an extent that an upward (as shown) force is produced on the spherical sealing pad 15 which overcomes both the attractive force between the magnet 16 and the magnet 14 and the weight of the valve assembly 8. If the upward force produced by the pressure differential equals or surpasses the magnetic force and the weight of the valve assembly 8 then the spherical seal pad 15 separates from the valve seat 10 to allow the passage of gas from the inlet 4 to the outlet 6.

The magnetic force between the spherical seal pad 15 and the valve seat 10 is adjusted to provide a force greater than the weight of valve assembly 8 such that when the pressure differential causes the spherical seal pad 15 to separate from the valve seat 10, thus cancelling the magnetic force, the valve assembly 8 is lifted clear. The gas flow between the flange 17 and the sleeve 3 produces an upward force greater than the weight of the valve assembly 8 propelling it towards the fully open position against the top cap 9.

With the valve 1 fully open there are three forces involved, namely the weight of the valve assembly 8, the force produced by the pressure differential across the recessed flange 17 and the magnetic force produced by the magnet 14 and the magnetic top cap 9. The magnetic force is adjusted to less than the weight of the

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valve assembly 8 by the relative positions of the magnet 14 and magnetic washer 13 to the valve top cap 9. This bridge could also be produced by incorporating an electro-magnet within or on the top cap 9.

When the force produced by the pressure differential between the flange 17 and the magnetic sleeve 3 is reduced the magnetic force cannot support the weight of the valve assembly 8 allowing it to lose contact with the top cap 9. The magnetic force between the valve assembly 8 and the top cap 9 becomes negligible as the valve assembly 8 falls downwards on the reduced gas glow until the closing magnetic force shuts the spherical seal pad 15 and the valve seat 10.

The pressure differential to initiate the opening by overcoming the magnetic force is greater than the pressure to move the valve assembly 8 to its fully open position. Similarly, the lower pressure differential required to initiate the closing by allowing the weight to overcome the magnetic force is lower than the pressure differential to keep the valve assembly 8 open and is not sufficient to support the weight of the valve assembly 8. This allows a free fall of the valve assembly 8 to the shut position.

The larger open and lesser closing pressure differentials are separated sufficiently to give a large hysteresis to prevent interactions between opening and closing as well as possible instability.

The shock absorber 11 provides a cushioning effect on the top surface of the valve stem assembly 8 to prevent or minimise valve flutter.

The polymer bush 7 protects the performance of the valve 1 in hostile environments by preventing deposition or corrosion on the sliding surfaces of the top cap 9 and the

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- stem 12 of the valve assembly 8. The main function of the magnetic sleeve 3 is to provide outlet ports and to keep the valve within predetermined boundaries. The fact that it is made from magnetic material prevents any lateral instability by biasing the valve assembly 8 towards the nearest point of the sleeve 3.
- The spherical seal pad 15 and the valve seat 10 geometry are arranged such that irrespective of the attitude or eccentricity of the valve assembly 8 in the magnetic sleeve 3 sealing is achieved by toppling onto the valve seat 10 without the necessity of laterally centralising by sliding.
 - The magnetic mounting aids 5 can be in the form of a wavy washer or polymer that absorbs any sudden shock or vibration thereby preventing damage to the permanent magnets 14, 16 which are usually brittle.

A particular advantage of the above described valve 1 is that the said valve opens at a predetermined pressure differential and remains open with a much lower pressure because once the valve assembly 8 has moved away from the valve seat 10 the magnetic force is reduced significantly. Furthermore, the combination of magnets, magnetic and non-magnetic materials along with the weight of the valve assembly 8 allows the user to tune easily the operating range. This is accomplished by selecting the relative position of the permanent magnets 14, 16 and their contact area which forms the actual sealing surface between the valve assembly 8 and the valve seat 10.

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WE CLAIM:

- 1. A valve comprising a housing having an inlet and space therefrom an outlet, a passageway extending between the inlet and the outlet, and means located in the passageway for controlling the flow of a fluid between the inlet and the outlet, the means including a valve assembly movable between a first open position spaced from a co-operating valve seat and a second closed position at which the valve assembly sealingly engages the valve seat, in which magnetic means is provided for biasing the valve stem assembly towards the second closed position.
- 2. The valve as claimed in Claim 1 in which at least a portion of the valve assembly is in the form of or incorporates a permanent magnet and a further magnet is located adjacent the valve seat.
- 3. The valve as claimed in Claim 2 in which the further magnet is a permanent magnet.
- 4. The valve as claimed in Claim 2 in which the further magnet is an electromagnet.
- 5. The valve as claimed in Claim 1 in which the valve seat is made of magnetic material.
- 6. The valve as claimed in Claim 1 in which the valve assembly includes a spherical seal pad which sealingly engages the valve seat in the second closed position of the valve assembly.

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- 7. The valve as claimed in Claim 1 in which the valve assembly depends from a valve cap made from magnetic material, which valve cap is sealingly attached to the housing.
- 8. The valve as claimed in Claim 7 in which an electro-magnet is uncorporated within or on the valve cap.
- 9. The valve as claimed in Claim 7 in which a magnetic sleeve depends from the valve cap and surrounds the valve assembly.
- 10. The valve as claimed in Claim 7 in which the valve assembly includes a shock absorber.
- 11. The valve as claimed in Claim 9 in which a polymer bush is provided which surrounds that portion of the valve assembly within the valve cap.

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ABSTRACT

A valve comprising a housing having an inlet and space therefrom an outlet, a passageway extending between the inlet and the outlet, and means located in the passageway for controlling the flow of a fluid between the inlet and the outlet, the means including a valve assembly movable between a first open position spaced from a co-operating valve seat and a second closed position at which the valve assembly sealingly engages the valve seat, in which magnetic means is provided for biasing the valve stem assembly towards the second closed position.

